POLYTECHNIC UNIVERSITY OF CATALONIA

MSC. IN NUMERICAL METHODS

## Computational Structural Mechanics and Dynamics Assignment 3.1:"The Plane Stress Problem" Assignment 3.2: "The 3-node Plane Stress Triangle"

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## Assignment 3.1: "The Plane Stress Problem"

(a)

For the plane stress problem, we call  $E = E^*$  and  $\nu = \nu^*$ . Making the stresses  $\sigma_{xx}$  and  $\sigma_{yy}$  equal for both plane stress and plane strain, we obtain the following system of equations

$$\frac{E(1-\nu)}{(1+\nu)(1-2\nu)} \left[ e_{xx} + \frac{\nu}{1-\nu} e_{yy} \right] = \frac{E*}{1-\nu^{*2}} \left[ \nu * e_{xx} + e_{yy} \right]$$
$$\frac{E(1-\nu)}{(1+\nu)(1-2\nu)} \left[ e_{yy} + \frac{\nu}{1-\nu} e_{xx} \right] = \frac{E*}{1-\nu^{*2}} \left[ \nu * e_{yy} + e_{xx} \right]$$

Solving for E\* and  $\nu*$  we obtain

$$E* = \frac{E}{1 - \nu^2}$$
$$\nu* = \frac{\nu}{1 - \nu}$$

Now, it is possible to go from one state to the other using this two fictitious modulus.

From plane stress

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix} = \frac{E*}{1 - \nu *^2} \begin{bmatrix} 1 & \nu * & 0 \\ \nu * & 1 & 0 \\ 0 & 0 & \frac{1 - \nu *}{2} \end{bmatrix} \begin{bmatrix} e_{xx} \\ e_{yy} \\ 2e_{xy} \end{bmatrix}$$

Replacing  $E* = \frac{E}{1-\nu^2}$  and  $\nu* = \frac{\nu}{1-\nu}$ 

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix} = \frac{\frac{E}{1-\nu^2}}{(1-(\frac{\nu}{1-\nu})^2)} \begin{bmatrix} 1 & \frac{\nu}{1-\nu} & 0 \\ \frac{\nu}{1-\nu} & 1 & 0 \\ 0 & 0 & \frac{1-\frac{\nu}{1-\nu}}{2} \end{bmatrix} \begin{bmatrix} e_{xx} \\ e_{yy} \\ 2e_{xy} \end{bmatrix}$$

Rearranging the fractions we end up with the plane strain system

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix} = \frac{E(1-\nu)}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1 & \frac{\nu}{1-\nu} & 0 \\ \frac{\nu}{1-\nu} & 1 & 0 \\ 0 & 0 & \frac{1-2\nu}{2(1-\nu)} \end{bmatrix} \begin{bmatrix} e_{xx} \\ e_{yy} \\ 2e_{xy} \end{bmatrix}$$

Then, to go back to plane stress we just have to invert the previous results  $E^* = \frac{E(2\nu+1)}{(\nu+1)^2}$  and  $\nu^* = \frac{\nu}{\nu+1}$ 

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix} = \frac{\frac{E(2\nu+1)}{(\nu+1)^2} (1-\frac{\nu}{\nu+1})}{(1+\frac{\nu}{\nu+1})(1-2\frac{\nu}{\nu+1})} \begin{bmatrix} 1 & \frac{\frac{\nu}{\nu+1}}{1-\frac{\nu}{\nu+1}} & 0 \\ \frac{\frac{\nu}{\nu+1}}{1-\frac{\nu}{\nu+1}} & 1 & 0 \\ 0 & 0 & \frac{1-2\frac{\nu}{\nu+1}}{2(1-\frac{\nu}{\nu+1})} \end{bmatrix} \begin{bmatrix} e_{xx} \\ e_{yy} \\ 2e_{xy} \end{bmatrix}$$

When simplified, the expression is

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix} = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix} \begin{bmatrix} e_{xx} \\ e_{yy} \\ 2e_{xy} \end{bmatrix}$$

(b)

Following Hook's law  $\sigma=eE$  and its inverse  $e=E^{-1}\sigma$  Then

$$U = \frac{1}{2}e^t Ee = \frac{1}{2}\sigma^t e = \frac{1}{2}(\sigma_{xx}e_{xx} + \sigma_{yy}e_{yy}\sigma_{xy}e_{xy})$$
$$U = \frac{1}{2}\sigma^t E^{-1}\sigma = \frac{1}{2}\sigma^t e = \frac{1}{2}(\sigma_{xx}e_{xx} + \sigma_{yy}e_{yy}\sigma_{xy}e_{xy})$$

## Assignment 3.2: "The 3-node Plane Stress Triangle"

(1)

Given the coordinates of the three points of the triangle

$$x_1=0 \ , \ x_2=3 \ , \ x_3=2 \ , \ y_1=0 \ , \ y_2=1 \ , \ y_3=2$$

We can calculate the area of the triangle by the following determinant

$$A = \frac{1}{2} \begin{vmatrix} 1 & x_1 & y_1 \\ 1 & x_2 & y_2 \\ 1 & x_3 & y_3 \end{vmatrix} = \frac{1}{2} \begin{vmatrix} 1 & 0 & 0 \\ 1 & 3 & 1 \\ 1 & 2 & 2 \end{vmatrix} = 2$$

And the element strain matrix

$$B = \frac{1}{2A} \begin{bmatrix} y_2 - y_3 & 0 & y_3 - y_1 & 0 & y_1 - y_2 & 0 \\ 0 & x_3 - x_2 & 0 & x_1 - x_3 & 0 & x_2 - x_1 \\ x_3 - x_2 & y_2 - y_3 & x_1 - x_3 & y_3 - y_1 & x_2 - x_1 & y_1 - y_2 \end{bmatrix} = \frac{1}{4} \begin{bmatrix} -1 & 0 & 2 & 0 & -1 & 0 \\ 0 & -1 & 0 & -2 & 0 & 3 \\ -1 & -1 & -2 & 2 & 3 & -1 \end{bmatrix}$$

For a constant thickness h = 1, the element stiffness matrix can be calculated as  $K_e = hB^T EB$ 

$$K_{e} = \frac{1}{8} \begin{bmatrix} -1 & 0 & -1 \\ -1 & -1 & 0 \\ -2 & 0 & 2 \\ 2 & -2 & 0 \\ 3 & 0 & -1 \\ -1 & 3 & 0 \end{bmatrix} \begin{bmatrix} 100 & 25 & 0 \\ 25 & 100 & 0 \\ 0 & 0 & 50 \end{bmatrix} \begin{bmatrix} -1 & 0 & 2 & 0 & -1 & 0 \\ 0 & -1 & 0 & -2 & 0 & 3 \\ -1 & -1 & -2 & 2 & 3 & -1 \end{bmatrix}$$
$$Ke = \begin{bmatrix} 18.75 & 9.375 & -12.5 & -6.25 & -6.25 & -3.125 \\ 9.375 & 18.75 & 6.25 & 12.5 & -15.625 & -31.25 \\ -12.5 & 6.25 & 75 & -37.5 & -62.5 & 31.25 \\ -6.25 & 12.5 & -37.5 & 75 & 43.75 & -87.5 \\ -6.25 & -15.625 & -62.5 & 43.75 & 68.75 & -28.125 \\ -3.125 & -31.25 & 31.25 & -87.5 & -28.125 & 118.75 \end{bmatrix}$$

(2)

The sum of rows (or columns) are

$$\begin{split} R_1 &= 18.75 + 9.375 - 12.5 - 6.25 - 6.25 + -3.125 = 0 \\ R_2 &= 9.375 + 18.75 + 6.25 + 12.5 - 15.625 - 31.25 = 0 \\ R_3 &= -12.5 + 6.25 + 75 - 37.5 - 62.5 + 31.25 = 0 \\ R_4 &= -6.25 + 12.5 - 37.5 + 75 + 43.75 - 87.5 = 0 \\ R_5 &= -6.25 - 15.625 - 62.5 + 43.75 + 68.75 - 28.125 = 0 \\ R_6 &= -3.125 - 31.25 + 31.25 - 87.5 - 28.125 + 118.75 = 0 \end{split}$$

The fact that the sum of R1 + R3 + R5 and R2 + R4 + R6 is equal to zero can be explained by the equilibrium of the forces in the triangle.

For the x coordinate

$$K_{11}u_{x1} + K_{12}u_{x2} + K_{13}u_{x3} + K_{14}u_{x4} + K_{15}u_{x5} + K_{16}u_{x6} = f_{x1}$$
  

$$K_{31}u_{x1} + K_{32}u_{x2} + K_{33}u_{x3} + K_{34}u_{x4} + K_{35}u_{x5} + K_{36}u_{x6} = f_{x2}$$
  

$$K_{51}u_{x1} + K_{52}u_{x2} + K_{53}u_{x3} + K_{54}u_{x4} + K_{55}u_{x5} + K_{56}u_{x6} = f_{x3}$$

Summing the tree equations, we can apply equilibrium of forces in x direction to be zero

$$\begin{split} K_{11}u_{x1} + K_{12}u_{x2} + K_{13}u_{x3} + K_{14}u_{x4} + K_{15}u_{x5} + K_{16}u_{x6} + \\ K_{31}u_{x1} + K_{32}u_{x2} + K_{33}u_{x3} + K_{34}u_{x4} + K_{35}u_{x5} + K_{36}u_{x6} + \\ K_{51}u_{x1} + K_{52}u_{x2} + K_{53}u_{x3} + K_{54}u_{x4} + K_{55}u_{x5} + K_{56}u_{x6} \\ &= f_{x1} + f_{x2} + f_{x3} = 0 \end{split}$$

It is evident that for any rigid displacement (because the non-existence of boundary conditions the resulting force should be zero, then the sum of the rigidity terms should also be zero.

The same explanations is valid for the y direction and columns 2,4,6.