

Assignment 3.1

STUDENTI:
Marcello Rubino

PAGE 1

$$1) \quad \lambda = \frac{E\nu}{(1+\nu)(1-2\nu)} \quad \mu = G = \frac{E}{2(1+\nu)}$$

$$\begin{cases} E\nu = (1+\nu)(1-2\nu)\lambda \\ E = 2(1+\nu)\mu \end{cases}$$

$$\begin{cases} 2\nu(1+\nu)\mu = (1+\nu)(1-2\nu)\lambda \\ \leftarrow \end{cases} \quad \begin{cases} 2\nu\mu = \lambda - 2\nu\lambda \\ \leftarrow \end{cases}$$

$$\begin{cases} 2\nu(\mu + \lambda) = \lambda \\ \leftarrow \end{cases} \quad \begin{cases} \nu = \frac{\lambda}{2(\mu + \lambda)} \\ E = 2\left(1 + \frac{\lambda}{2(\mu + \lambda)}\right)\mu \end{cases} \quad \begin{cases} \leftarrow \\ E = 2 \cdot \left(\frac{2(\mu + \lambda) + \lambda}{2(\mu + \lambda)}\right)\mu \end{cases}$$

$$E = \frac{(2\mu + 3\lambda)\mu}{\mu + \lambda} \quad \nu = \frac{\lambda}{2(\mu + \lambda)}$$

2) PLANE STRESS \underline{E} MATRIX:

$$\underline{E} = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix} \begin{matrix} xx \\ yy \\ xy \end{matrix}$$

$$\underline{E} = \frac{4\mu(2\mu^2 + 5\lambda\mu + 3\lambda^2)}{4\mu^2 + 8\lambda\mu + 3\lambda^2} \cdot \begin{bmatrix} 1 & \frac{\lambda}{2(\mu + \lambda)} & 0 \\ \frac{\lambda}{2(\mu + \lambda)} & 1 & 0 \\ 0 & 0 & \frac{2\mu + \lambda}{4(\mu + \lambda)} \end{bmatrix}$$

PLANE STRAIN \underline{E} MATRIX

$$\underline{E} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & 0 \\ \nu & 1-\nu & 0 \\ 0 & 0 & \frac{1}{2}(1-2\nu) \end{bmatrix}$$

$$\underline{E} = \begin{bmatrix} 2\mu+1 & 1 & 0 \\ 1 & 2\mu+1 & 0 \\ 0 & 0 & \mu \end{bmatrix}$$

$$3) \quad \underline{E}_\mu + \underline{E}_\lambda = \underbrace{\begin{bmatrix} 2\mu & 0 & 0 \\ 0 & 2\mu & 0 \\ 0 & 0 & \mu \end{bmatrix}}_{\underline{E}_\mu} + \underbrace{\begin{bmatrix} 1 & 1 & 0 \\ 1 & 1 & 0 \\ 0 & 0 & 0 \end{bmatrix}}_{\underline{E}_\lambda}$$

$$4) \quad \underline{E}_\mu = \begin{bmatrix} 2\mu & 0 & 0 \\ 0 & 2\mu & 0 \\ 0 & 0 & \mu \end{bmatrix} = \begin{bmatrix} \frac{E}{1+\nu} & 0 & 0 \\ 0 & \frac{E}{1+\nu} & 0 \\ 0 & 0 & \frac{E}{2(1+\nu)} \end{bmatrix}$$

$$\underline{E}_\lambda = \begin{bmatrix} 1 & 1 & 0 \\ 1 & 1 & 0 \\ 0 & 0 & 0 \end{bmatrix} = \begin{bmatrix} \frac{E\nu}{(1+\nu)(1-2\nu)} & \frac{E\nu}{(1+\nu)(1-2\nu)} & 0 \\ \frac{E\nu}{(1+\nu)(1-2\nu)} & \frac{E\nu}{(1+\nu)(1-2\nu)} & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

Assignment 3.2

We consider the plane stress case, and initially $\nu = 0$.

1) - (a)

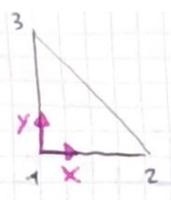
$$\underline{E} = \begin{bmatrix} E & 0 & 0 \\ 0 & E & 0 \\ 0 & 0 & E/2 \end{bmatrix} \quad \underline{B} = \frac{1}{2A} \begin{bmatrix} Y_{23} & 0 & Y_{31} & 0 & Y_{12} & 0 \\ 0 & X_{32} & 0 & X_{13} & 0 & X_{21} \\ X_{32} & Y_{23} & X_{13} & Y_{31} & X_{21} & Y_{12} \end{bmatrix}$$

$$\text{where: } Y_{ij} = Y_i - Y_j$$

$$X_{ij} = X_i - X_j$$

in this case:

$$\underline{B} = \frac{1}{2A} \begin{bmatrix} -a & 0 & a & 0 & 0 & 0 \\ 0 & -a & 0 & 0 & 0 & a \\ -a & -a & 0 & a & a & 0 \end{bmatrix}$$



$h = \text{constant}$, so:

$$K_{tri}^e = A h \underline{B}^T \underline{E} \underline{B} = \frac{h}{4A} \begin{bmatrix} -a & 0 & -a \\ 0 & -a & -a \\ a & 0 & 0 \\ 0 & 0 & a \\ 0 & 0 & a \\ 0 & a & 0 \end{bmatrix} \begin{bmatrix} E & 0 & 0 \\ 0 & E & 0 \\ 0 & 0 & E/2 \end{bmatrix} \begin{bmatrix} -a & 0 & a & 0 & 0 & 0 \\ 0 & -a & 0 & 0 & 0 & a \\ -a & -a & 0 & a & a & 0 \end{bmatrix}$$

$$= \frac{h}{4A} \begin{bmatrix} -aE & 0 & -aE/2 \\ 0 & -aE & -aE/2 \\ aE & 0 & 0 \\ 0 & 0 & aE/2 \\ 0 & 0 & aE/2 \\ 0 & aE & 0 \end{bmatrix} \begin{bmatrix} \vdots \\ \vdots \\ \vdots \\ \vdots \\ \vdots \\ \vdots \end{bmatrix} =$$

$$K_{tri}^e = \frac{h}{2a^2} \begin{bmatrix} \frac{3a^2E}{2} & \frac{a^2E}{2} & -a^2E & -\frac{a^2E}{2} & -\frac{a^2E}{2} & 0 \\ \frac{a^2E}{2} & \frac{3a^2E}{2} & 0 & -\frac{a^2E}{2} & -\frac{a^2E}{2} & -a^2E \\ -a^2E & 0 & a^2E & 0 & 0 & 0 \\ -\frac{a^2E}{2} & -\frac{a^2E}{2} & 0 & \frac{a^2E}{2} & \frac{a^2E}{2} & 0 \\ -\frac{a^2E}{2} & -\frac{a^2E}{2} & 0 & \frac{a^2E}{2} & \frac{a^2E}{2} & 0 \\ 0 & -a^2E & 0 & 0 & 0 & a^2E \end{bmatrix} = \begin{bmatrix} \frac{3Eh}{4} & \frac{Eh}{4} & -\frac{Eh}{2} & -\frac{Eh}{4} & -\frac{Eh}{4} & 0 \\ \frac{Eh}{4} & \frac{3Eh}{4} & 0 & -\frac{Eh}{4} & -\frac{Eh}{4} & \frac{Eh}{2} \\ -\frac{Eh}{2} & 0 & \frac{Eh}{2} & 0 & 0 & 0 \\ -\frac{Eh}{4} & -\frac{Eh}{4} & 0 & \frac{Eh}{4} & \frac{Eh}{4} & 0 \\ -\frac{Eh}{4} & -\frac{Eh}{4} & 0 & \frac{Eh}{4} & \frac{Eh}{4} & 0 \\ 0 & -\frac{Eh}{2} & 0 & 0 & 0 & \frac{Eh}{2} \end{bmatrix}$$

1) - (b)

$$\frac{1}{K_{(1)}} = \begin{bmatrix} \frac{EA_1}{a} & 0 & -\frac{EA_1}{a} & 0 \\ 0 & 0 & 0 & 0 \\ -\frac{EA_1}{a} & 0 & \frac{EA_1}{a} & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \rightarrow \frac{1}{K_{(1)}} = \begin{bmatrix} 0 & 0 & 0 & 0 \\ 0 & \frac{EA_1}{a} & 0 & -\frac{EA_1}{a} \\ 0 & 0 & 0 & 0 \\ 0 & -\frac{EA_1}{a} & 0 & \frac{EA_1}{a} \end{bmatrix}$$

3-1

$$\tilde{\underline{K}}_{(2)}^e = \underline{K}_{(2)}^e = \begin{bmatrix} \frac{EA_2}{a} & 0 & -\frac{EA_2}{a} & 0 \\ 0 & 0 & 0 & 0 \\ -\frac{EA_2}{a} & 0 & \frac{EA_2}{a} & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

1-2

$$\tilde{\underline{K}}_{(3)}^e = \begin{bmatrix} \frac{EA_3}{\sqrt{2}a} & 0 & -\frac{EA_3}{\sqrt{2}a} & 0 \\ 0 & 0 & 0 & 0 \\ \frac{EA_3}{\sqrt{2}a} & 0 & \frac{EA_3}{\sqrt{2}a} & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \rightarrow \underline{K}_{(3)}^e = \begin{bmatrix} \frac{EA_3\sqrt{2}}{4a} & -\frac{EA_3\sqrt{2}}{4a} & -\frac{EA_3\sqrt{2}}{4a} & \frac{EA_3\sqrt{2}}{4a} \\ -\frac{EA_3\sqrt{2}}{4a} & \frac{EA_3\sqrt{2}}{4a} & \frac{EA_3\sqrt{2}}{4a} & -\frac{EA_3\sqrt{2}}{4a} \\ -\frac{EA_3\sqrt{2}}{4a} & \frac{EA_3\sqrt{2}}{4a} & \frac{EA_3\sqrt{2}}{4a} & -\frac{EA_3\sqrt{2}}{4a} \\ \frac{EA_3\sqrt{2}}{4a} & -\frac{EA_3\sqrt{2}}{4a} & -\frac{EA_3\sqrt{2}}{4a} & \frac{EA_3\sqrt{2}}{4a} \end{bmatrix}$$

2-3

↓ Knowing that $A_1 = A_2 = A$

$$\underline{K}_{\text{bar}}^e = \begin{bmatrix} \frac{EA}{a} & 0 & -\frac{EA}{a} & 0 & 0 & 0 \\ 0 & \frac{EA}{a} & 0 & 0 & 0 & -\frac{EA}{a} \\ -\frac{EA}{a} & 0 & \frac{EA + EA_3\sqrt{2}}{4a} & -\frac{EA_3\sqrt{2}}{4a} & -\frac{EA\sqrt{2}}{4a} & \frac{EA_3\sqrt{2}}{4a} \\ 0 & 0 & -\frac{EA_3\sqrt{2}}{4a} & \frac{EA_3\sqrt{2}}{4a} & \frac{EA_3\sqrt{2}}{4a} & -\frac{EA_3\sqrt{2}}{4a} \\ 0 & 0 & -\frac{EA_3\sqrt{2}}{4a} & \frac{EA_3\sqrt{2}}{4a} & \frac{EA_3\sqrt{2}}{4a} & -\frac{EA_3\sqrt{2}}{4a} \\ 0 & -\frac{EA}{a} & \frac{EA_3\sqrt{2}}{4a} & -\frac{EA_3\sqrt{2}}{4a} & -\frac{EA_3\sqrt{2}}{4a} & \frac{EA + EA_3\sqrt{2}}{4a} \end{bmatrix}$$

- 2) Reading the two matrices we can see that there is no possible set of values for $A_1 = A_2 = A$ and A_3 to make them equivalent. We can only see similarities in the elements 1-1, 2-2, only if $\frac{A}{a} = \frac{3h}{4}$; in elements 3-3 and 6-6 or in 4-4, 4-5, 5-4, 5-5; where there is equivalence if $\frac{A_3\sqrt{2}}{4a} = \frac{h}{4}$. The rest of the matrices are completely different, because for some do the bars structure doesn't bring any internal force while the triangle does. ($\underline{K}_{\text{tri}}$ is more "powerful" meaning that the possible internal power is much greater). If the solution is linear than it's really difficult or impossible to use tubular structures of bars because they don't bring any contribution in some degrees of freedom.

4) Considering now considering $v \neq 0$, we have that:

$$K_{tt}^e = Ah \cdot \underline{B}^T \underline{E} \underline{B} = \frac{h}{4A} \begin{bmatrix} -a & 0 & -a \\ 0 & -a & -a \\ a & 0 & 0 \\ 0 & 0 & a \\ 0 & 0 & a \\ 0 & a & 0 \end{bmatrix} \begin{bmatrix} 1 & v & 0 \\ v & 1 & 0 \\ 0 & 0 & \frac{1-v}{2} \end{bmatrix} \cdot \frac{E}{1-\nu^2} \cdot \underline{B}$$

$$= \frac{Eh}{2a^2(1-\nu^2)} \begin{bmatrix} -a & -a\nu & -\frac{a(1-\nu)}{2} \\ -a\nu & -a & -\frac{a(1-\nu)}{2} \\ a & a\nu & \frac{a(1-\nu)}{2} \\ 0 & 0 & \frac{a(1-\nu)}{2} \\ 0 & 0 & \frac{a(1-\nu)}{2} \\ a\nu & a & 0 \end{bmatrix} \begin{bmatrix} -a & 0 & a & 0 & 0 & 0 \\ 0 & -a & 0 & 0 & 0 & a \\ -a & -a & 0 & a & a & 0 \end{bmatrix} =$$

$$= \frac{Eh}{2a^2(1-\nu^2)} \begin{bmatrix} a^2(1 + \frac{1-\nu}{2}) & a^2(\nu + \frac{1-\nu}{2}) & -a^2 & -a^2 \frac{1-\nu}{2} & -a^2 \frac{1-\nu}{2} & -a^2 \nu \\ a^2(\nu + \frac{1-\nu}{2}) & a^2(1 + \frac{1-\nu}{2}) & -a^2 \nu & -a^2 \frac{1-\nu}{2} & -a^2 \frac{1-\nu}{2} & -a^2 \\ -a^2 & -a^2 \nu & a^2 & 0 & 0 & a^2 \nu \\ -a^2 \cdot \frac{1-\nu}{2} & -a^2 \cdot \frac{1-\nu}{2} & 0 & a^2 \frac{1-\nu}{2} & a^2 \frac{1-\nu}{2} & 0 \\ -a^2 \cdot \frac{1-\nu}{2} & -a^2 \cdot \frac{1-\nu}{2} & 0 & a^2 \frac{1-\nu}{2} & a^2 \frac{1-\nu}{2} & 0 \\ -a^2 \nu & -a^2 & a^2 \nu & 0 & 0 & a^2 \end{bmatrix} =$$

$$= \frac{Eh}{(1-\nu^2)} \begin{bmatrix} \frac{3-\nu}{4} & \frac{1+\nu}{4} & -\frac{1}{2} & -\frac{1-\nu}{4} & -\frac{1-\nu}{4} & -\frac{\nu}{2} \\ \frac{1+\nu}{4} & \frac{3-\nu}{4} & -\frac{\nu}{2} & -\frac{1-\nu}{4} & -\frac{1-\nu}{4} & -\frac{1}{2} \\ -\frac{1}{2} & -\frac{\nu}{2} & \frac{1}{2} & 0 & 0 & \frac{\nu}{2} \\ -\frac{1-\nu}{4} & -\frac{1-\nu}{4} & 0 & \frac{1-\nu}{4} & \frac{1-\nu}{4} & 0 \\ -\frac{1-\nu}{4} & -\frac{1-\nu}{4} & 0 & \frac{1-\nu}{4} & \frac{1-\nu}{4} & 0 \\ -\frac{\nu}{2} & -\frac{1}{2} & \frac{\nu}{2} & 0 & 0 & \frac{1}{2} \end{bmatrix} =$$

$$K_{tri}^e = \begin{bmatrix} \frac{(3-\nu)Eh}{4(1-\nu^2)} & \frac{Eh}{4(1-\nu)} & -\frac{Eh}{2(1-\nu^2)} & -\frac{Eh}{4(1+\nu)} & -\frac{Eh}{4(1+\nu)} & -\frac{Eh\nu}{2(1-\nu^2)} \\ \frac{Eh}{4(1-\nu)} & \frac{(3-\nu)Eh}{4(1-\nu^2)} & -\frac{Eh\nu}{2(1-\nu^2)} & \frac{Eh}{4(1+\nu)} & -\frac{Eh}{4(1+\nu)} & -\frac{Eh}{2(1-\nu^2)} \\ \frac{Eh}{2(1-\nu^2)} & -\frac{Eh\nu}{2(1-\nu^2)} & \frac{Eh}{2(1-\nu^2)} & 0 & 0 & \frac{Eh\nu}{2(1-\nu^2)} \\ -\frac{Eh}{4(1+\nu)} & -\frac{Eh}{4(1+\nu)} & 0 & \frac{Eh}{4(1+\nu)} & \frac{Eh}{4(1+\nu)} & 0 \\ -\frac{Eh}{4(1+\nu)} & -\frac{Eh}{4(1+\nu)} & 0 & \frac{Eh}{4(1+\nu)} & \frac{Eh}{4(1+\nu)} & 0 \\ -\frac{Eh\nu}{2(1-\nu^2)} & -\frac{Eh}{2(1-\nu^2)} & \frac{Eh\nu}{2(1-\nu^2)} & 0 & 0 & \frac{Eh}{2(1-\nu^2)} \end{bmatrix}$$

In this case the difference with the bar structure is more evident. There are more non-zero elements than before, so that means that this model of structure is more precise, because there are more components that can give us a non-zero force. About the elements similar to the K_{bar}^e we can make the same consideration, but changing the relations obviously:

$$\bullet \frac{A}{a} = \frac{(3-\nu)h}{4(1-\nu^2)} \quad \bullet \frac{A_3\sqrt{2}}{4a} = \frac{h}{4(1-\nu^2)}$$

Also in this case we can see that the response of the structure due to "for" dof is completely different and more accurate than the bar structure: the "mixed" elements are 0 mostly in the K_{bar}^e while in the triangle are non-zero values.