# COMPUTATIONAL SUCTURAL MECHANICS AND DYNAMICS <br> Master of Science in Computational Mechanics/Numerical Methods <br> Spring Semester 2019 

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## Assignment 2: FEM Modelling: Introduction

1. Identify the symmetry and antisymmetry lines in the two-dimensional problems illustrated in the figure. They are:

Symmetry lines are marked with red lines while antisymmetry lines are marked in blue.
(a)


(c)


(e)
2. Having identified those symmetry/antisymmetry lines, state whether it is possible to cut the complete structure to one half or one quarter before laying out a finite element mesh. Then draw a coarse FE mesh indicating, with rollers or fixed supports, which kind of displacement BCs you would specify on the symmetry or antisymmetry lines.
(a)

(d)

(c)

(f)


Here, in the a), b) and e) cases, the force to apply is $\mathrm{P} / 2$.
3. The plate structure shown in the figure is loaded and deforms in the plane of the paper. The applied load at $D$ and the supports $I$ and $N$ extend over a fairly narrow area. List what you think are the likely "trouble spots" that would require a locally finer element mesh to capture high stress gradients. Identify those spots by its letter and a reason.

The stress concentration may be due to two causes:

- Point loads or loads applied on a narrow area: This includes the points $D, N$ and $I$.
- Non-convex domains: For that reason, the mesh should be refined around B, F, M and J.


4. A tapered bar element of length $I$ and areas $\boldsymbol{A}_{i}$ and $\boldsymbol{A}_{\boldsymbol{j}}$ with $\boldsymbol{A}$ interpolated as:

$$
A=A_{i}(1-\xi)+A_{j} \xi
$$

And constant density $\rho$ rotates on a plane at uniform angular velocity $\omega$ ( $\mathrm{rad} / \mathrm{sec}$ ) about node $\boldsymbol{i}$. Taking axis $\mathbf{x}$ along the rotating bar with origin at node $I$, the centrifugal axial force is $q(x)=\rho A \omega^{2} x$ along the length in which $\mathbf{x}$ is the longitudinal coordinate $x=x^{e}$.
Find the consistent node forces as functions of $\rho, \boldsymbol{A}_{\boldsymbol{i}}, \boldsymbol{A}_{\boldsymbol{j}}, \boldsymbol{\omega}$ and $I$, and specialize the result to the prismatic bar $A=A_{i}=A_{j}$.

First of all, the $\xi$ coordinate is expressed in terms of the element coordinate $x$ :

$$
\xi=\frac{x}{l}
$$

Now, the unknown variables are expressed in function of $\xi$ :

$$
\boldsymbol{N}=\left[\begin{array}{c}
1-\xi \\
\xi
\end{array}\right]
$$

The force vector is calculated as follows:

$$
\begin{gathered}
\boldsymbol{f}=\int_{0}^{1} \boldsymbol{N} \cdot q \cdot J^{-1} d \xi=\int_{0}^{1}\left[\begin{array}{c}
1-\xi \\
\xi
\end{array}\right] \rho\left(A_{i}(1-\xi)+A_{j} \xi\right) \omega^{2} \xi \cdot l \cdot l d \xi \\
=\rho \omega^{2} l^{2} \int_{0}^{1} A_{i}\left[\begin{array}{c}
1-\xi)^{2} \xi \\
\xi^{2}-\xi^{3}
\end{array}\right]+A_{j}\left[\begin{array}{c}
\xi^{2}-\xi^{3} \\
\xi^{3}
\end{array}\right] d \xi \\
=\rho \omega^{2} l^{2}\left(A_{i}\left(\left[\begin{array}{c}
1 / 12 \\
1 / 12
\end{array}\right]\right)+A_{j}\left(\left[\begin{array}{c}
1 / 12 \\
1 / 4
\end{array}\right]\right)\right)
\end{gathered}
$$

For the case of the prismatic bar:

$$
\boldsymbol{f}=\rho \omega^{2} l^{2} A\left(\left[\begin{array}{l}
1 / 6 \\
1 / 3
\end{array}\right]\right)
$$

