

Assignment 3.1

$$\lambda = \frac{Ev}{(1+v)(1-2v)} \quad \text{--- (1)}; \quad \mu = \frac{E}{2(1+v)} \Rightarrow E = 2\mu(1+v) \quad \text{--- (2)}$$

from substituting μ (2) in (1) we get,

$$\lambda = \frac{2(1+v)v\mu}{(1+v)(1-2v)}$$

$$\lambda = \frac{2v\mu}{1-2v}$$

$$\Rightarrow \frac{\mu}{\lambda} = \frac{1-2v}{2v}$$

$$\Rightarrow \frac{\mu}{\lambda} = \frac{1}{2v} - 1$$

$$\Rightarrow \frac{\mu}{\lambda} + 1 = \frac{1}{2v}$$

$$\Rightarrow \frac{(\mu + \lambda)}{\lambda} = \frac{1}{2v}$$

$$\Rightarrow \boxed{v = \frac{\lambda}{2(\mu + \lambda)}} \quad \text{--- (3)}$$

from (2) we have,

$$E = 2\mu(1+v)$$

$$= 2\mu \left(1 + \frac{\lambda}{2(\mu + \lambda)} \right)$$

$$= 2\mu \left(\frac{2\mu + 2\lambda + \lambda}{2(\mu + \lambda)} \right)$$

$$\therefore E = \mu \left(\frac{2\mu + 3\lambda}{\mu + \lambda} \right) \quad \text{--- (4)}$$

2. Plane Stress.

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix} = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix} \begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{xy} \end{bmatrix}$$

where $\frac{E}{1-\nu^2} = \frac{E}{(1+\nu)(1-\nu)} \quad \text{--- (5)}$

Substituting values of (3) & (4) we get

$$\begin{aligned} \frac{E}{(1+\nu)(1-\nu)} &= \frac{\mu \left(\frac{2\mu + 3\lambda}{\mu + \lambda} \right)}{\left(1 + \frac{\lambda}{2(\mu + \lambda)} \right) \left(1 - \frac{\lambda}{2(\mu + \lambda)} \right)} \\ &= \frac{\mu \left(\frac{2\mu + 3\lambda}{\mu + \lambda} \right)}{\frac{(2\mu + 2\lambda + \lambda)(2\mu + \lambda)}{4(\mu + \lambda)^2}} = \frac{\mu \left(\frac{2\mu + 3\lambda}{\mu + \lambda} \right)}{\frac{(2\mu + 3\lambda)(2\mu + \lambda)}{4(\mu + \lambda)^2}} \\ &\Rightarrow \boxed{\frac{E}{1-\nu^2} = \frac{4\mu(\mu + \lambda)}{2\mu + \lambda}} \quad \text{--- (6)} \end{aligned}$$

$$\frac{1-\nu}{2} = \frac{1 - \frac{\lambda}{2(\mu + \lambda)}}{2}$$

$$= \frac{2\mu + 2\lambda - \lambda}{4(\mu + \lambda)} \Rightarrow \boxed{\frac{1-\nu}{2} = \frac{2\mu + \lambda}{4(\mu + \lambda)}}$$

∴ From Plane Stress eqⁿ for 2D problem we

have,

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix} = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix} \begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{xy} \end{bmatrix}$$

$$\Rightarrow \begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix} = \frac{4\mu(\mu+\lambda)}{2\mu+\lambda} \begin{bmatrix} 1 & \frac{\lambda}{2(\mu+\lambda)} & 0 \\ \frac{\lambda}{2(\mu+\lambda)} & 1 & 0 \\ 0 & 0 & \frac{2\mu+\lambda}{4(\mu+\lambda)} \end{bmatrix} \begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{xy} \end{bmatrix}$$

2. Plane Strain

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix} = \frac{E(1-\nu)}{(1+\nu)(1+2\nu)} \begin{bmatrix} 1 & \frac{\nu}{1-\nu} & 0 \\ \frac{\nu}{1-\nu} & 1 & 0 \\ 0 & 0 & \frac{1-2\nu}{2(1-\nu)} \end{bmatrix} \begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{xy} \end{bmatrix}$$

where,

$$\frac{E(1-\nu)}{(1+\nu)(1-2\nu)} = \frac{\mu \left(\frac{2\mu+3\lambda}{\mu+\lambda} \right) \left(1 - \frac{\lambda}{2(\mu+\lambda)} \right)}{\left(1 + \frac{\lambda}{2(\mu+\lambda)} \right) \left(1 - \frac{2\lambda}{2(\mu+\lambda)} \right)}$$

$$= \frac{\mu \left(\frac{2\mu+3\lambda}{\mu+\lambda} \right) \left(\frac{2\mu+2\lambda-\lambda}{2(\mu+\lambda)} \right)}{\left(\frac{2\mu+2\lambda+\lambda}{2(\mu+\lambda)} \right) \left(\frac{2\mu+2\lambda-2\lambda}{2(\mu+\lambda)} \right)} = \frac{\mu(2\mu+\lambda)}{\mu}$$

$$\therefore \frac{E(1-\nu)}{(1+\nu)(1-2\nu)} = 2\mu + \lambda$$

$$\text{and, } \frac{\nu}{1-\nu} = \frac{\lambda}{2(\mu + \lambda)}$$

$$= \frac{\lambda}{\cancel{2(\mu + \lambda)}} \cdot \frac{\cancel{2\mu + 2\lambda - \lambda}}{2(\mu + \lambda)}$$

$$\boxed{\frac{\nu}{1-\nu} = \frac{\lambda}{2\mu + \lambda}}$$

$$\text{and, } \frac{1-2\nu}{2(1-\nu)} = \frac{1 - \frac{2\lambda}{2(\mu + \lambda)}}{2\left(\frac{2\mu + \lambda}{2(\mu + \lambda)}\right)}$$

$$= \frac{2\mu}{2(\mu + \lambda)} \times \frac{(\mu + \lambda)}{2\mu + \lambda}$$

$$= \frac{\mu}{2\mu + \lambda}$$

$$\begin{bmatrix} b_{xx} \\ b_{yy} \\ b_{xy} \end{bmatrix} = (2\mu + \lambda) \begin{bmatrix} 1 & \frac{\lambda}{2\mu + \lambda} & 0 \\ \frac{2\lambda}{2\mu + \lambda} & 1 & 0 \\ 0 & 0 & \frac{\mu}{2\mu + \lambda} \end{bmatrix} \begin{bmatrix} e_{xx} \\ e_{yy} \\ e_{xy} \end{bmatrix}$$

The Stress Strain Matrix E

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix} = \begin{bmatrix} 2\mu + \lambda & \lambda & 0 \\ \lambda & 2\mu + \lambda & 0 \\ 0 & 0 & \mu \end{bmatrix} \begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{xy} \end{bmatrix}$$

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix} = \underbrace{\begin{bmatrix} 2\mu & 0 & 0 \\ 0 & 2\mu & 0 \\ 0 & 0 & \mu \end{bmatrix}}_{E_{\mu}} + \underbrace{\begin{bmatrix} \lambda & \lambda & 0 \\ \lambda & \lambda & 0 \\ 0 & 0 & 0 \end{bmatrix}}_{E_{\lambda}} \begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{xy} \end{bmatrix}$$

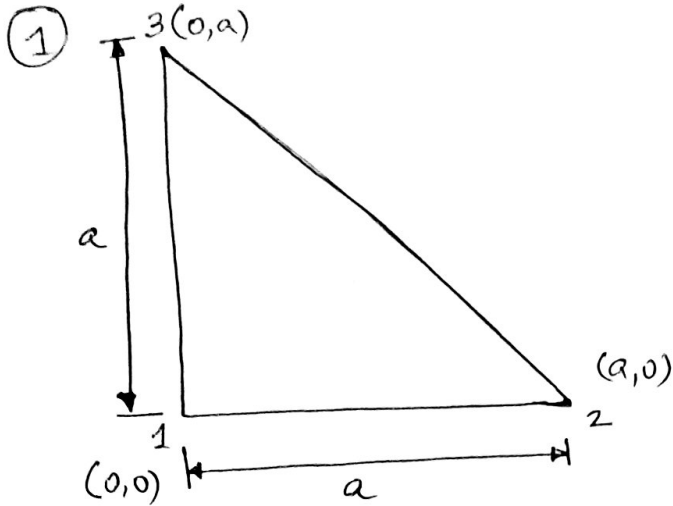
$$\{\sigma\} = \left([E_{\mu}] + [E_{\lambda}] \right) \{e\}$$

4) E_{λ} in terms of E & ν

$$E_{\lambda} = \frac{E\nu}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1 & 1 & 0 \\ 1 & 1 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

$$E_{\mu} = \frac{E}{2(1+\nu)} \begin{bmatrix} 2 & 0 & 0 \\ 0 & 2 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

Assignment 3.2



$$K_{tri} = A h B^T I B$$

$$= \frac{h}{4A} \begin{bmatrix} y_{23} & 0 & x_{32} \\ 0 & x_{32} & y_{22} \\ y_{31} & 0 & x_{13} \\ 0 & x_{13} & y_{31} \\ y_{12} & 0 & x_{21} \\ 0 & x_{21} & y_{12} \end{bmatrix} \times \begin{bmatrix} E_{11} & E_{12} & E_{13} \\ E_{12} & E_{22} & E_{23} \\ E_{13} & E_{23} & E_{33} \end{bmatrix}$$

$$x_1 = 0, y_1 = 0$$

$$x_2 = a, y_2 = 0$$

$$x_3 = 0, y_3 = a$$

$$y_{23} = y_2 - y_3 = 0 - a = -a; \quad x_{32} = x_3 - x_2 = -a$$

$$y_{31} = y_3 - y_1 = a; \quad x_{13} = x_1 - x_3 = 0$$

$$y_{12} = y_1 - y_2 = 0; \quad x_{21} = x_2 - x_1 = a$$

$$\times \begin{bmatrix} y_{23} & 0 & y_{31} & 0 & y_{12} & 0 \\ 0 & x_{32} & 0 & x_{13} & 0 & x_{21} \\ x_{32} & y_{23} & x_{13} & y_{13} & x_{21} & y_{21} \end{bmatrix}$$

$$\therefore K_{tri} = \frac{Eh}{4A(1-\nu^2)} \begin{bmatrix} -a & 0 & -a \\ 0 & -a & -a \\ a & 0 & 0 \\ 0 & 0 & a \\ 0 & 0 & a \\ 0 & a & 0 \end{bmatrix} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix} \begin{bmatrix} -a & 0 & a & 0 & 0 & a \\ 0 & -a & 0 & 0 & 0 & a \\ -a & -a & 0 & a & a & 0 \end{bmatrix}$$

$$= \frac{Eh}{4A(1-\nu^2)} \begin{bmatrix} -a & 0 & -a \\ 0 & -a & -a \\ a & 0 & 0 \\ 0 & 0 & a \\ 0 & 0 & a \\ 0 & a & 0 \end{bmatrix} \begin{bmatrix} -a & -\nu a & a & 0 & 0 & \nu a \\ -\nu a & -a & \nu a & 0 & 0 & a \\ -\frac{a+\nu a}{2} & \frac{-a+\nu a}{2} & 0 & \frac{a-\nu a}{2} & \frac{a-\nu a}{2} & 0 \end{bmatrix}$$

$$= \frac{Eh}{2a^2(1-v^2)}$$

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$a^2 + \frac{a^2 - va^2}{2}$	$va^2 + \frac{a^2 - va^2}{2}$	$-a^2$	$-\frac{a^2 + va^2}{2}$	$-\frac{a + va^2}{2}$	$-va^2$
$va^2 + \frac{a^2 - va^2}{2}$	$a^2 + \frac{a^2 - va^2}{2}$	$-va^2$	$-\frac{a^2 + va^2}{2}$	$-\frac{a + va^2}{2}$	$-a^2$
$-a^2$	$-va^2$	a^2	0	0	va^2
$-\frac{a^2 + va^2}{2}$	$-\frac{a^2 + va^2}{2}$	0	$\frac{a^2 - va^2}{2}$	$\frac{a^2 - va^2}{2}$	0
$-\frac{a^2 + va^2}{2}$	$-\frac{a^2 + va^2}{2}$	0	$\frac{a^2 - va^2}{2}$	$\frac{a^2 - va^2}{2}$	0
$-va^2$	$-a^2$	va^2	0	0	a^2

$$= \frac{Eh}{2(1-v^2)}$$

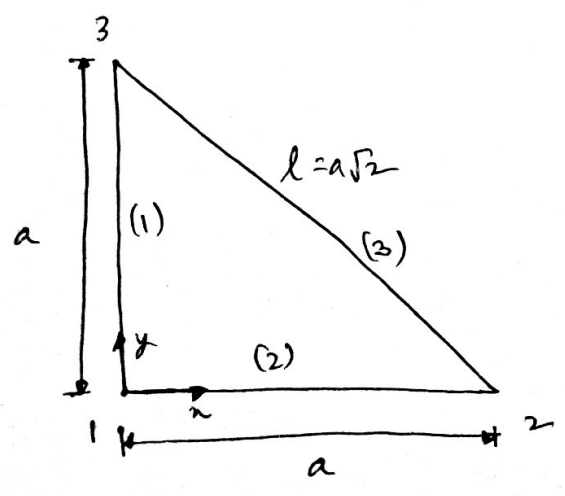
$1 + \frac{1-v}{2}$	$v + \frac{1-v}{2}$	-1	$-\frac{1+v}{2}$	$-\frac{1+v}{2}$	$-v$
$v + \frac{1-v}{2}$	$1 + \frac{1-v}{2}$	$-v$	$-\frac{1+v}{2}$	$-\frac{1+v}{2}$	-1
-1	$-v$	1	0	0	v
$-\frac{1+v}{2}$	$-\frac{1+v}{2}$	0	$\frac{1-v}{2}$	$\frac{1-v}{2}$	0
$-\frac{1+v}{2}$	$-\frac{1+v}{2}$	0	$\frac{1-v}{2}$	$\frac{1-v}{2}$	0
$-v$	-1	v	0	0	1

② ~~Part~~

Taking $a=1, h=1$ & $\nu=0$, we have,

$$K_{Tri} = \frac{E}{2} \begin{bmatrix} \frac{3}{2} & \frac{1}{2} & -1 & -\frac{1}{2} & -\frac{1}{2} & 0 \\ \frac{1}{2} & \frac{3}{2} & 0 & -\frac{1}{2} & -\frac{1}{2} & -1 \\ -1 & 0 & 1 & 0 & 0 & 0 \\ -\frac{1}{2} & -\frac{1}{2} & 0 & \frac{1}{2} & \frac{1}{2} & 0 \\ -\frac{1}{2} & -\frac{1}{2} & 0 & \frac{1}{2} & \frac{1}{2} & 0 \\ 0 & -1 & 0 & 0 & 0 & 1 \end{bmatrix}$$

For K_{bar}



For element (2)

$$K^{(2)} = \frac{A_2 E}{a} \begin{bmatrix} c_2^2 & c_2 s_2 & -c_2^2 & -c_2 s_2 \\ c_2 s_2 & s_2^2 & -c_2 s_2 & -s_2^2 \\ -c_2^2 & -c_2 s_2 & c_2^2 & c_2 s_2 \\ -c_2 s_2 & -s_2^2 & c_2 s_2 & s_2^2 \end{bmatrix}$$

where $c_2 = \cos 0^\circ = 1$
 $s_2 = \sin 0^\circ = 0$

Hence, $K^{(2)} = \frac{A_2 E}{a}$

$$\begin{bmatrix} 1 & 0 & -1 & 0 \\ 0 & 0 & 0 & 0 \\ -1 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

For Element 1.

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$$\theta = 90^\circ, \quad c_1 = \cos \theta = \cos 90^\circ = 0 \\ s_1 = \sin \theta = \sin 90^\circ = 1$$

$$K^{(1)} = \frac{A_1 E}{a} \begin{bmatrix} c_1^2 & c_1 s_1 & -c_1^2 & -c_1 s_1 \\ c_1 s_1 & s_1^2 & -c_1 s_1 & -s_1^2 \\ -c_1^2 & -c_1 s_1 & c_1^2 & c_1 s_1 \\ -c_1 s_1 & -s_1^2 & c_1 s_1 & s_1^2 \end{bmatrix}$$

$$K^{(1)} = \frac{A_1 E}{a} \begin{bmatrix} 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & -1 \\ 0 & 0 & 0 & 0 \\ 0 & -1 & 0 & 1 \end{bmatrix}$$

For element 3

$$\theta = 135^\circ$$

$$c_3 = \cos 135^\circ = -\frac{1}{\sqrt{2}}$$

$$s_3 = \sin 135^\circ = \frac{1}{\sqrt{2}}$$

$$c_3^2 = \frac{1}{2}; \quad -c_3^2 = -\frac{1}{2}$$

$$c_3 s_3 = -\frac{1}{2}; \quad -c_3 s_3 = \frac{1}{2}$$

$$s_3^2 = \frac{1}{2}; \quad -s_3^2 = -\frac{1}{2}$$

$$K^{(3)} = \frac{A_3 E}{a\sqrt{2}} \begin{bmatrix} c_3^2 & c_3 s_3 & -c_3^2 & -c_3 s_3 \\ c_3 s_3 & s_3^2 & -c_3 s_3 & -s_3^2 \\ -c_3^2 & -c_3 s_3 & c_3^2 & c_3 s_3 \\ -c_3 s_3 & -s_3^2 & c_3 s_3 & s_3^2 \end{bmatrix}$$

$$K^{(3)} = \frac{A_3 E}{a \sqrt{2}} \begin{bmatrix} \frac{1}{2} & -\frac{1}{2} & -\frac{1}{2} & \frac{1}{2} \\ -\frac{1}{2} & \frac{1}{2} & \frac{1}{2} & -\frac{1}{2} \\ -\frac{1}{2} & \frac{1}{2} & \frac{1}{2} & -\frac{1}{2} \\ \frac{1}{2} & -\frac{1}{2} & -\frac{1}{2} & \frac{1}{2} \end{bmatrix}$$

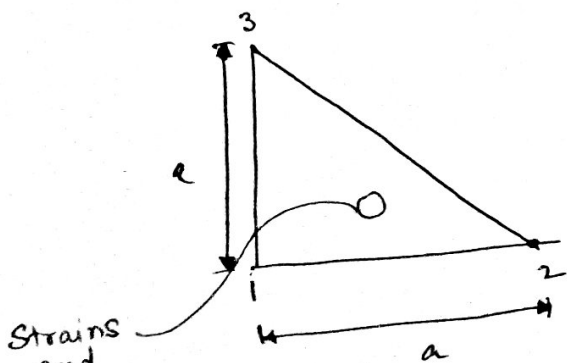
$$K_{\text{bar}} = \begin{bmatrix} \frac{A_2 E}{a} & 0 & -\frac{A_2 E}{a} & 0 & 0 & 0 \\ 0 & \frac{A_1 E}{a} & 0 & 0 & 0 & -\frac{A_1 E}{a} \\ -\frac{A_2 E}{a} & 0 & \frac{A_2 E}{a} + \frac{A_3 E}{a 2\sqrt{2}} & -\frac{A_3 E}{a 2\sqrt{2}} & -\frac{A_3 E}{2\sqrt{2}a} & \frac{A_3 E}{2\sqrt{2}a} \\ 0 & 0 & -\frac{A_3 E}{a 2\sqrt{2}} & \frac{A_3 E}{a 2\sqrt{2}} & \frac{A_3 E}{2\sqrt{2}a} & -\frac{A_3 E}{2\sqrt{2}a} \\ 0 & 0 & -\frac{A_3 E}{a 2\sqrt{2}} & \frac{A_3 E}{2\sqrt{2}a} & \frac{A_3 E}{2\sqrt{2}a} & -\frac{A_3 E}{2\sqrt{2}a} \\ 0 & -\frac{A_1 E}{a} & \frac{A_3 E}{2\sqrt{2}a} & -\frac{A_3 E}{2\sqrt{2}a} & -\frac{A_3 E}{2\sqrt{2}a} & \frac{A_1 E}{a} + \frac{A_3 E}{2\sqrt{2}a} \end{bmatrix}$$

2. There are no values of cross section for which ~~$A_1 = A_2$~~ both the stiffness matrices are equivalent. i.e. $K_{bar} = K_{Tri}$

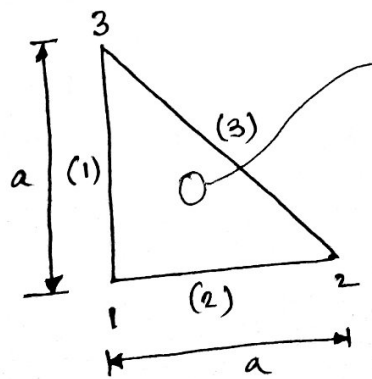
Values with $A_3 = \sqrt{2}a$ and $A_1 = A_2 = \frac{a}{2}$ may make the matrices more similar, but not equal

3. Where as the $(K_{bar})_{bar}$ matrices are specifically formulated to take care of Trusses, Turner Triangle matrices are formulated to take care of stresses in Plane Stress problems.

The Turner Triangle is not able to take into account the strains ^{somewhere} in the middle of the boundary of the element. That is not possible for the Bar (K_{bar}) element



Strains and displacements taken into account due to the type of Shape function considered



Strains and displacements not taken into account due to the Shape function taking into account displacement only along the axis of the each bar element

④ For $\nu \neq 0$

we will have

$$K_{Tri} = \frac{E}{2(1-\nu^2)}$$

$$\begin{bmatrix}
 1 + \frac{1-\nu}{2} & \nu + \frac{1-\nu}{2} & -1 & -\frac{1+\nu}{2} & -\frac{1+\nu}{2} & -\nu \\
 \nu + \frac{1-\nu}{2} & 1 + \frac{1-\nu}{2} & -\nu & -\frac{1+\nu}{2} & -\frac{1+\nu}{2} & -1 \\
 -1 & -\nu & 1 & 0 & 0 & \nu \\
 -\frac{1+\nu}{2} & -\frac{1+\nu}{2} & 0 & \frac{1-\nu}{2} & \frac{1-\nu}{2} & 0 \\
 -\frac{1+\nu}{2} & -\frac{1+\nu}{2} & 0 & \frac{1-\nu}{2} & \frac{1-\nu}{2} & 0 \\
 -\nu & -1 & \nu & 0 & 0 & 1
 \end{bmatrix}$$

When $\nu \neq 0$, the Element matrix takes into account the lateral displacement that may take place which is not the case when $\nu = 0$.
~~where~~