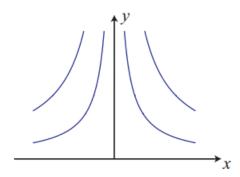
Homework 4.

Task 1.

Consider a fluid stream whose velocity vector coincides with the y axis that impinges on a plane boundary that coincides with the x axis, as shown in the figure below.



(a) Stream function $\Psi(r,\theta) = Ur^2 \sin(2\theta) = 2Ur^2 \sin(\theta) \cos(\theta)$. As going to Cartesian coordinates $x = r \cos(\theta)$ and $y = r \sin(\theta)$, the stream function takes the form $\Psi(x,y) = 2Uxy$. Let us compute the velocity field:

$$\begin{cases} u = \frac{\partial \Psi(x, y)}{\partial y} = 2Ux\\ v = -\frac{\partial \Psi(x, y)}{\partial x} = -2Uy \end{cases}$$

For ideal fluid the point (0, 0) is a stagnation point, so, here the velocity should be 0. Indeed u(0,0)=v(0,0)=0.

From Bernoulli's equation we receive the pressure distribution:

$$\frac{1}{2}\boldsymbol{v}^2 + \frac{p}{\rho} = \frac{p_0}{\rho} \quad \Rightarrow \quad p = p_0 - \frac{\rho}{2}\boldsymbol{v}^2 \quad \Rightarrow \quad p = p_0 - 2\rho U^2 (x^2 + y^2)$$

where $\boldsymbol{v} = \begin{pmatrix} u(x,y) \\ v(x,y) \end{pmatrix}$.

(b) As we consider ideal fluid, the velocity does not depend on time ($v_t = 0$) and there are no body forces, the Navier-Stokes equations take the view:

$$\begin{cases} \nabla \cdot \boldsymbol{v} = 0\\ (\boldsymbol{v} \cdot \nabla) \boldsymbol{v} - \boldsymbol{v} \nabla^2 \boldsymbol{v} + \frac{1}{\rho} \nabla p = 0 \end{cases}$$

Let us check the first equation:

$$\nabla \cdot \boldsymbol{v} = \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 2U - 2U = 0$$

The second equation has the form:

$$\begin{cases} (\boldsymbol{v} \cdot \boldsymbol{\nabla})u - v\boldsymbol{\nabla}^2 u + \frac{1}{\rho}\frac{\partial p}{\partial x} = 0\\ (\boldsymbol{v} \cdot \boldsymbol{\nabla})v - v\boldsymbol{\nabla}^2 v + \frac{1}{\rho}\frac{\partial p}{\partial y} = 0 \end{cases}$$

Applying the velocity field to these equations, we receive:

$$u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial y} - v\left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2}\right) + \frac{1}{\rho}\frac{\partial p}{\partial x} = 4U^2x + 0 - 0 - 4U^2x = 0;$$
$$u\frac{\partial v}{\partial x} + v\frac{\partial v}{\partial y} - v\left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2}\right) + \frac{1}{\rho}\frac{\partial p}{\partial y} = 4U^2y + 0 - 0 - 4U^2y = 0.$$

Which means that the obtained velocity satisfies to Navier-Stokes equations.

For viscous fluid there is a no-slip boundary condition at wall for the velocity which means that v = 0 if y=0. This condition cannot be satisfied as velocity component u does not depend on y and u(x, 0) = 2Ux.

(c) Let us attempt the *u* velocity component for viscous fluid as u = 2Uxf'(y). As $u = \frac{\partial \Psi(x,y)}{\partial y}$, we can find the stream function: $\Psi(x,y) = \int u dy = \int 2Uxf'(y) dy = 2Uxf(y)$. Now we can obtain the velocity component $v: v = -\frac{\partial \Psi(x,y)}{\partial x} = -2Uf(y)$.

Let us determine boundary conditions for the function f(y). As v(x, 0) = 0, f(0) = 0. And as u(x, 0) = 0, f'(y) = 0.

For region sufficiently away from the wall, the viscous effect is negligible and the flow is expected to match with the inviscid flow result. Thus we require:

$$\begin{cases} 2Uxf'(y) \to 2Ux \\ -2Uf(y) \to -2Uy \end{cases} \Rightarrow \begin{cases} f'(y) \to 1 \\ f(y) \to y \end{cases} \text{ when } y \to \infty \end{cases}$$

To sum up, boundary conditions for f:

$$f(0) = f'(y) = 0$$

$$\begin{cases} f'(y) \to 1 \\ f(y) \to y \end{cases} \text{ when } y \to \infty$$

(d) Let us consider the y-momentum equation to get pressure distribution in terms of function f(y):

$$u\frac{\partial v}{\partial x} + v\frac{\partial v}{\partial y} - v\left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2}\right) + \frac{1}{\rho}\frac{\partial p}{\partial y} = 4U^2f(y)f'(y) + v \cdot 2Uf''(y) + \frac{1}{\rho}\frac{\partial p}{\partial y} = 0$$
$$\frac{\partial p}{\partial y} = -\rho\left(4U^2f(y)f'(y) + v \cdot 2Uf''(y)\right)$$

Integrating, we receive:

$$p = -\rho(2U^2f^2 + 2\nu Uf') + C(x)$$

Recalling that $f(y) \rightarrow y$ for large values of y shows that

$$p \to -\rho(2U^2y^2 + 2\nu U) + C(x)$$

which by comparison with the potential-flow pressure, requires:

$$p_0 - 2\rho U^2(x^2 + y^2) = -\rho(2U^2y^2 + 2\nu U) + C(x) \Rightarrow C(x) = p_0 - 2\rho U^2x^2 + 2\rho\nu U.$$

Substituting, we receive the following pressure distribution:

$$p = -2\rho U^2 f^2 - 2\rho v U f' + p_0 - 2\rho U^2 x^2 + 2\rho v U = p_0 - 2\rho U^2 (f^2 + x^2) + 2\rho v U (1 - f').$$

(e) Let us consider the x-momentum equation and obtained formula for pressure distribution:

$$u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial y} - v\left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2}\right) + \frac{1}{\rho}\frac{\partial p}{\partial x} = 4U^2 x(f')^2 - 4U^2 x f'' - 2vUx f''' - 4U^2 x = 0 \quad \Rightarrow \\ \frac{v}{2U}f''' - (f')^2 + ff'' + 1 = 0.$$

We have received differential equation for function f. Let us introduce the dimensionless variables:

$$\eta = y \sqrt{\frac{2U}{v}}$$
 and $F(\eta) = \sqrt{\frac{2U}{v}} f(y)$.

Then the equation takes the view:

$$F^{'''} - (F^{'})^2 + FF^{''} + 1 = 0$$

where $F' = \frac{\partial F}{\partial \eta}$. Applying received boundary conditions for the function f(y), receive BC for new function F(η):

$$F(0) = F'(0) = 0; F'(\eta) \rightarrow 1 \text{ when } \eta \rightarrow \infty$$

This third-order ODE can be solved numerically.

Task 2.

A quadratic polynomial form for the velocity profile:

$$\frac{u}{U} = a + b\left(\frac{y}{\delta}\right) + c\left(\frac{y}{\delta}\right)^2$$

Boundary conditions:

$$u = 0 \quad if \ y = 0$$
$$\begin{cases} u = U \\ \frac{\partial u}{\partial y} = 0 \quad if \ y = \delta \end{cases}$$

Applying boundary conditions, we receive the system of equations:

$$\begin{cases} \frac{u}{U}(y=0) = a = 0;\\ \frac{u}{U}(y=\delta) = a + b + c = 1;\\ \frac{\left(\frac{\partial u}{\partial y}\right)}{U}(y=\delta) = b + 2c = 0. \end{cases}$$

Solving the system, receive unknown parameters: $\begin{cases} a = 0 \\ b = 2 \\ c = -1 \end{cases}$

These conditions give:

$$\frac{u}{U} = 2\left(\frac{y}{\delta}\right) - \left(\frac{y}{\delta}\right)^2.$$

The momentum integral equation reduces to:

$$\frac{d}{dx}(U^2\theta) = \frac{\tau_0}{\rho} \quad \Rightarrow \quad \frac{d}{dx} \int_0^\infty u(U-u) \, dy = \frac{\tau_0}{\rho} \qquad (1)$$

where $U^2\theta = \int_0^\infty u(U-u) \, dy$ - momentum thickness.

Thus, we can compute the momentum thickness:

$$\theta = U^{-2} \int_{0}^{\delta} u(U-u) \, dy = \int_{0}^{\delta} \left(\frac{u}{U} - \left(\frac{u}{U}\right)^{2}\right) \, dy$$
$$= \int_{0}^{\delta} \left(2\left(\frac{y}{\delta}\right) - \left(\frac{y}{\delta}\right)^{2} - 4\left(\frac{y}{\delta}\right)^{2} - \left(\frac{y}{\delta}\right)^{4} + 4\left(\frac{y}{\delta}\right)^{3}\right) \, dy = \frac{2}{15}\delta \qquad (2)$$

The shear stress on the surface is given by formula:

$$\tau_0 = \mu \frac{\partial u}{\partial y}|_{y=0} = \mu \frac{\partial u}{\partial y}|_{y=0} = \mu \left(\frac{2}{\delta} - \frac{2y}{\delta^2}\right) U|_{y=0} = \frac{2U\mu}{\delta}.$$

As
$$\mu = \rho \nu \Rightarrow \frac{\tau_0}{\rho} = \frac{2U\nu}{\delta}$$
 (3).

From equations (1)-(3) we receive:

$$\frac{2}{15}U^2\frac{d\delta}{dx} = \frac{2U\nu}{\delta} \quad \Rightarrow \quad \delta d\delta = \frac{15\nu}{U}dx \quad \Rightarrow \quad \delta^2 = \frac{30\nu}{U}x + C.$$

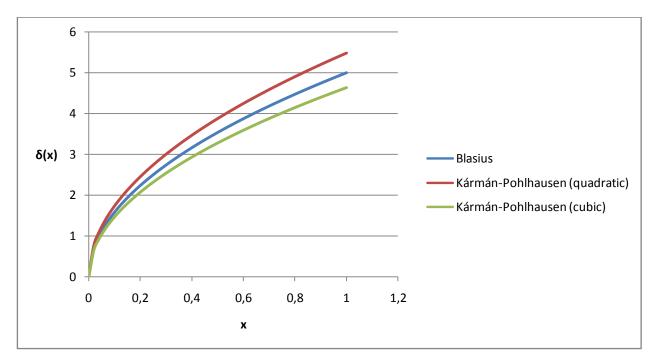
Assuming $\delta(0) = 0$, we receive $\delta = 5.477 \sqrt{\frac{vx}{v}} = 5.477 \frac{x}{\sqrt{Re_x}}$ where $Re_x = \frac{Ux}{v}$.

From (2) receive the momentum thickness: $\theta = \frac{2}{15} \delta = 0.7303 \frac{x}{\sqrt{Re_x}}$.

Comparing the results with the exact Blasius solution and with the ones obtained assuming a cubic velocity profile:

	$\frac{u}{U}$	δ	θ
Blasius	$f'\left(\frac{y}{\sqrt{\frac{yx}{U}}}\right)$	$5\frac{x}{\sqrt{Re_x}}$	$0.664 \frac{x}{\sqrt{Re_x}}$
Kármán-Pohlhausen (quadratic)	$2\left(\frac{y}{\delta}\right) - \left(\frac{y}{\delta}\right)^2$	$5.477 \frac{x}{\sqrt{Re_x}}$	$0.7303 \frac{x}{\sqrt{Re_x}}$
Kármán-Pohlhausen (cubic)	$\frac{3}{2} \left(\frac{y}{\delta}\right) - \frac{1}{2} \left(\frac{y}{\delta}\right)^2$	$4.64 \frac{x}{\sqrt{Re_x}}$	$0.646 \frac{x}{\sqrt{Re_x}}$

The results of the boundary layer thickness $\delta(x)$:



The results of the momentum thickness $\theta(x)$:

