

ADVANCED FLUID MECHANICS : HOMEWORK 4

- ④ a) Ideal fluid, stream function:

$$\Psi(r, \theta) = Ur^2 \cdot \sin(2\theta)$$

- Velocity field in cartesian coordinates (u, v) and showing bc. verification.

- Expression for the pressure distribution.

$$\text{Velocity field in polar} \left\{ \begin{array}{l} U_r = \frac{1}{r} \frac{\partial \Psi}{\partial \theta} \rightarrow U_r = 2Ur \cdot \cos(2\theta) \\ U_\theta = -\frac{\partial \Psi}{\partial r} \rightarrow U_\theta = -2Ur \cdot \sin(2\theta). \end{array} \right.$$

$$\text{Velocity field in cartesian} \left\{ \begin{array}{l} u = U_r \cdot \cos \theta - U_\theta \cdot \sin \theta \\ v = U_r \cdot \sin \theta + U_\theta \cdot \cos \theta \\ x = r \cdot \cos \theta, \quad y = r \cdot \sin \theta, \quad r = \sqrt{x^2 + y^2} \end{array} \right.$$

$$\left\{ \begin{array}{l} u = 2Ur \cdot \cos 2\theta \cdot \cos \theta + 2Ur \cdot \sin 2\theta \cdot \sin \theta \\ v = 2Ur \cdot \cos 2\theta \cdot \sin \theta - 2Ur \cdot \sin 2\theta \cdot \cos \theta \\ x = r \cdot \cos \theta \\ y = r \cdot \sin \theta \end{array} \right. \rightarrow \left\{ \begin{array}{l} u = 2U \cos 2\theta x + 2U \sin 2\theta y \\ v = 2U \cos 2\theta y - 2U \sin 2\theta x \end{array} \right.$$

$$\begin{aligned} \cos(2\theta) &= \cos^2 \theta - \sin^2 \theta \\ \sin(2\theta) &= 2 \sin \theta \cdot \cos \theta \\ \cos^2 \theta + \sin^2 \theta &= 1 \end{aligned}$$

$$\rightarrow \left\{ \begin{array}{l} u = 2Ux(2\cos^2 \theta - 1) + 2Uy(2\sin \theta \cos \theta) \\ v = -2Ux(2\sin \theta \cos \theta) + 2Uy(2\cos^2 \theta - 1) \\ u = -2Ux + 4U \cos \theta (\cos \theta x + \sin \theta y) \\ v = -2Uy + 4U \cos \theta (\cos \theta y - \sin \theta x) \end{array} \right.$$

$$\cos \theta = \frac{x}{r} = \frac{x}{\sqrt{x^2+y^2}}, \quad \sin \theta = \frac{y}{r} = \frac{y}{\sqrt{x^2+y^2}}$$

$$u = -2U_x + 4U \left(\frac{x}{\sqrt{x^2+y^2}} \right) \left(\frac{x^2}{\sqrt{x^2+y^2}} + \frac{y^2}{\sqrt{x^2+y^2}} \right) = 2U_x$$

$$v = -2U_y + 4U \left(\frac{y}{\sqrt{x^2+y^2}} \right) \left(\frac{xy}{\sqrt{x^2+y^2}} + \frac{xy}{\sqrt{x^2+y^2}} \right) = -2U_y$$

* Velocity field (Cartesian coordinates) $\Rightarrow \boxed{\underline{V} = [2U_x, -2U_y]}$

* BC. verification $-\nabla \cdot \underline{V} = 0$ in stagnation point $(0,0) \rightarrow \underline{V}(0,0) = [0, 0] \checkmark$

- in $x=0$, v parallel to y -axis $\rightarrow \underline{V}(0,y) = (0, -2U_y) \checkmark$

- in flat plate, $v=0 \rightarrow \underline{V}(x,0) = [2U_x, 0] \checkmark$

* Pressure distribution

$$\omega = \begin{vmatrix} e_x & e_y & e_z \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ 2U_x & -2U_y & 0 \end{vmatrix} = \frac{\partial}{\partial x}(-2U_y)e_z + \frac{\partial}{\partial z}(2U_x)e_y = 0$$

In rotational velocity field.

~~stationary problem~~ ~~so the Bernoulli eq. becomes.~~

$$\int \frac{\partial \underline{V}}{\partial t} ds + \left(\frac{1}{2} \underline{V}_2^2 + \frac{P_2}{\rho} - F_2 \right) - \left(\frac{1}{2} \underline{V}_1^2 + \frac{P_1}{\rho} - F_1 \right) = 0$$

$$|\underline{V}| = \sqrt{(2U_x)^2 + (-2U_y)^2} = 2U \sqrt{x^2+y^2}$$

Taking 1 as stagnation point and 2 as any other point in the domain.

$$\left(\frac{1}{2} \left((2U \sqrt{x^2+y^2})^2 + \frac{P_2}{\rho} + g y_2 \right) - \left(0 + \frac{P_{max}}{\rho} + g y_1 \right) \right) = 0$$

$$2U^2(x^2+y^2) + \frac{P_2}{\rho} = \frac{P_{max}}{\rho}$$

$$\frac{P_{\max} - P_0}{\rho} = 2U^2(x^2 + y^2)$$

$$P(x,y) = P_{\max} - 2U^2 \rho (x^2 + y^2)$$

Pressure at the stagnation point.

b) Navier-Stokes eq:

$$\bullet \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \quad (\text{Mass conservation}).$$

$$\frac{\partial (2U_x)}{\partial x} + \frac{\partial (-2U_y)}{\partial y} = 0 \rightsquigarrow 2U - 2U = 0 \quad \checkmark$$

$$\bullet \rho \left(\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = - \frac{\partial p}{\partial x} + \mu \nabla^2 u + \rho b_x \quad (\text{x-momentum}).$$

STEADY STATE

$$\rho (2U_x) \frac{\partial (2U_x)}{\partial x} = - \frac{\partial (P_{\max} - 2U^2 \rho (x^2 + y^2))}{\partial x} + \mu \nabla^2 (2U_x).$$

$$\rho 4U_x^2 = 2U^2 \rho 2x + \mu \left(\frac{\partial^2 (2U_x)}{\partial x^2} + \frac{\partial^2 (2U_x)}{\partial y^2} + \frac{\partial^2 (2U_x)}{\partial z^2} \right)$$

$$\rho 4U_x^2 = 2U^2 \rho 2x \quad \checkmark$$

$$\bullet \rho \left(\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \right) = - \frac{\partial p}{\partial y} + \mu \nabla^2 v + \rho b_y \quad (\text{y-momentum}).$$

$$\rho (-2U_y) \frac{\partial (-2U_y)}{\partial y} = - \frac{\partial (P_{\max} - 2U^2 \rho (x^2 + y^2))}{\partial y} + \mu \nabla^2 (-2U_y).$$

$$\rho 4U_y^2 = 2U^2 \rho 2y + \mu \nabla^2 v$$

$$\rho 4U_y^2 = \rho 4U_y^2 \quad \checkmark$$

All the navier-Stokes equations are fulfilled

For viscous fluids velocity must be zero at the "wall" ($y=0$).

$\bullet u = 2U_x \rightsquigarrow$ if $x \neq 0$ and $v \neq 0 \rightarrow$ No slip conditions at wall

$\bullet v = 2U_y \rightsquigarrow$ if $y = 0$ and $v \neq 0 \rightarrow$ vertical bc is fulfilled.

c)

$$u = 2U_x f'(y)$$

$$v = -2U_y f(y)$$

State appropriate bc for the function "f"

$$(x^2 + y^2)^{1/2} \cos \theta = \frac{g - u}{v}$$

$$(x^2 + y^2)^{1/2} \cos \theta = \frac{g - u}{v} = \text{const}$$

$$V(x, 0) = 0 \rightarrow \text{Condition in the surface}$$

$V(x, y) = (2U_x, -2U_y) \rightarrow$ Potential flow velocity field for a point far from the surface.

$$0 = 2U_x f'(0) \rightsquigarrow f'(0) = 0$$

$$0 = -2U_y f(0) \rightsquigarrow f(0) = 0$$

$$2U_x = 2U_x f'(y) \rightsquigarrow f'(y) = 1$$

$$-2U_y = -2U_y f(y) \rightsquigarrow f(y) = y$$

d). \checkmark Y-momentum eq: $\rho \left(\frac{\partial v}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial v}{\partial y} \right) = - \frac{\partial p}{\partial y} + \mu V^2 + g b y$

$$\rho \left((-2U_x f'(y)) \frac{\partial (-2U_x f'(y))}{\partial x} + (-2U_y f(y)) \frac{\partial (-2U_y f(y))}{\partial x} \right) = - \frac{\partial p}{\partial y} + \mu V^2 (-2U_y f(y))$$

$$\rho \left(-2U_y f(y) \left(-2U_y \frac{\partial f(y)}{\partial y} \right) \right) = - \frac{\partial p}{\partial y} + \mu \left(\frac{\partial^2 (-2U_x f'(y))}{\partial x^2} + \frac{\partial^2 (-2U_y f(y))}{\partial y^2} \right)$$

$$\rho [4U^2 f'(y) f''(y)] = - \frac{\partial p}{\partial y} + \mu (-2U_y f''(y))$$

$$\int \frac{1}{\rho} \frac{\partial p}{\partial y} dy = \int -2U_y \mu f''(y) dy - 4U^2 f'(y) f''(y) dy$$

Integrating.

$$\frac{P(x, y)}{\rho} = -2U_y \mu f'(y) - 2U^2 (f'(y))^2 + P_0(x)$$

$$P(x, y) = -2U_y \mu f'(y) - 2\rho U^2 f'^2(y) + P_0(x)$$

If $y \rightarrow \infty$ the potential flow is recovered, so the pressure distribution must be the same that the one obtained in the exercise a).

$$\bullet P(x,y) = P_{\max} - 2U^2 \rho(x^2 + y^2)$$

$$\bullet \text{If } y \uparrow \rightarrow f(y) = 1$$

$$f(y) = y$$

$$P(x,y) = -2U\mu \frac{f(y)}{1} - 2\rho U^2 \frac{f^2(y)}{y^2} + P_0(x) = P_{\max} - 2U^2 \rho(x^2 + y^2).$$

$$-2U\mu - 2\rho U^2 y^2 + P_0(x) = P_{\max} - 2U^2 \rho(x^2 + y^2)$$

$$P_0(x) = P_{\max} - 2U^2 \rho x^2 + 2U\mu \rightarrow P_0(x) = P_{\max} + 2U(\mu - \rho p x^2).$$

$$P(x,y) = P_{\max} + 2U\mu(1-f(y)) - 2U^2 \rho(f^2(y) + x^2).$$

e) x-momentum eq: $\cancel{\rho} \left(\cancel{\frac{\partial u}{\partial t}} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = - \frac{\partial p}{\partial x} + \mu \sqrt{u} + \cancel{p b x}$

$$\bullet P(x,y) = P_{\max} + 2U\mu(1-f(y)) - 2U^2 \rho(f^2(y) + x^2)$$

$$\bullet \underline{U} = [2Ux f'(y), -2Uf(y)]$$

$$\cancel{\rho} \left[(2Ux f'(y)) \frac{\partial (2Ux f'(y))}{\partial x} + (2Uf(y)) \frac{\partial (2Ux f'(y))}{\partial y} \right] = - \frac{\partial p}{\partial x} + \mu \sqrt{2Ux f'(y)}.$$

$$\cancel{\rho} \left[4U^2 (f'(y))^2 - 4U^2 f'(y) f''(y) \right] = -(-4U^2 p x + 2Ux f'''(y)\mu)$$

$$\frac{1}{4U^2} \cdot (4U^2 p x (f'(y))^2 - 4U^2 p x f'(y) f''(y)) = (4U^2 p x + 2Ux f'''(y)\mu) \cdot \frac{1}{4U^2 p x}$$

$$(f'(y))^2 - f'(y) f''(y) - 1 - \frac{f'''(y)\mu}{2U} = 0$$

$$(f'(y))^2 - f'(y) f''(y) - 1 - \frac{f'''(y)\mu}{2U} = 0$$

$$\boxed{\frac{\mu}{2U} f'''(y) + f''(y) f'(y) - f'(y) f'(y) + 1 = 0} \quad \text{EDO}$$

BC.s:

$$\begin{aligned} f'(0) &= 0 & f'(y) &= 1 \\ f(0) &= 0 & f(y) &= y \quad \left\{ \begin{array}{l} (y \rightarrow \infty) \\ \end{array} \right. \end{aligned}$$

$$\frac{D}{2U} f'''(0) + \cancel{\frac{f''(0)}{2U} f(0)} - \cancel{\frac{f'(0)}{2U} f''(0)} + 1 = 0$$

$$\frac{D}{2U} f'''(0) + 1 = 0 \rightarrow \boxed{f'''(0) = -\frac{2U}{D}}$$

$$\frac{D}{2U} f'''(y) + \cancel{\frac{f''(y)}{2U} f(y)} - \cancel{\frac{f'(y)}{2U} f''(y)} + 1 = 0$$

$$\frac{D}{2U} f'''(y) - 1 + 1 = 0 \rightarrow \boxed{f'''(y) = 0}$$

② Boundary layer solution for an uniform flow over a flat plate:

- Blasius exact solution
- Kármán - Pohlhausen approx: * Quadratic
* Cubic

Kármán - Pohlhausen approximation (quadratic).

$$\frac{u}{U} = a + b \left(\frac{y}{\delta}\right) + c \left(\frac{y}{\delta}\right)^2 \quad \left\{ \begin{array}{l} u=0 \quad \text{for } y=0 \\ u=U \quad ; \quad \frac{\partial u}{\partial y} = 0 \quad \text{for } y=\delta \end{array} \right. \quad \left. \begin{array}{l} \frac{u}{U} = 0 \quad \text{for } y/\delta = 0 \\ \frac{u}{U} = 1 \quad ; \quad \frac{\partial(u/U)}{\partial(y/\delta)} = 0 \quad \text{for } y/\delta = 1. \end{array} \right.$$

$$0 = a + b \cdot 0 + c \cdot 0 \rightarrow a = 0$$

$$1 = b(1) + c(1)^2 \quad \left\{ \begin{array}{l} 1 = -2c + c \rightarrow c = -1 \\ b = 2 \end{array} \right. \rightarrow \frac{u}{U} = 2 \left(\frac{y}{\delta}\right) - \left(\frac{y}{\delta}\right)^2$$

$$0 = b + 2c(1)$$

Momentum integral equation (flat plate): $\frac{d}{dx} \int_0^{\delta} u(u-u) dy = \frac{\tau_0}{\rho}$

$$\frac{d}{dx} \int_0^{\delta} u \left(2\frac{y}{\delta} - \frac{y^2}{\delta^2} \right) \left(u - u \left(2\frac{y}{\delta} - \frac{y^2}{\delta^2} \right) \right) dy = \frac{\tau_0}{\rho} \quad \tau_0 = \mu \frac{U}{\delta} \left. \frac{\partial u}{\partial y} \right|_{y=\delta}$$

$$U^2 \frac{d}{dx} \int_0^{\delta} \left(\frac{2y}{\delta} - \frac{y^2}{\delta^2} \right) \left(1 - \frac{2y}{\delta} + \frac{y^2}{\delta^2} \right) dy = \frac{\tau_0}{\rho} \quad \tau_0 = \mu \frac{U}{\delta}$$

$$\frac{d}{dx} \int_0^{\delta} \left(\frac{2y}{\delta} - \frac{5y^2}{3\delta^2} + \frac{4y^3}{4\delta^3} - \frac{y^4}{5\delta^4} \right) dy = \frac{\tau_0}{\rho U}$$

$$\frac{d}{dx} \left[\frac{2y^2}{2\delta} - \frac{5y^3}{3\delta^2} + \frac{4y^4}{4\delta^3} - \frac{y^5}{5\delta^4} \right]_0^{\delta} = \frac{2\delta}{\rho U} \rightarrow \frac{d}{dx} \left(\frac{2\delta}{15} \right) = \frac{\tau_0}{\rho U}$$

$$\boxed{\frac{d}{dx} (\delta) = \frac{15\tau}{\rho U}} \rightarrow \text{ODE}$$

$$\delta \cdot d\delta = \frac{15\tau}{\rho U} dx \rightarrow \int \delta \cdot d\delta = \int \frac{15\tau}{\rho U} dx \rightarrow \frac{\delta^2}{2} = \frac{15\tau x}{\rho U} + C$$

$$\text{If } x=0 \rightarrow \delta=0 \rightarrow \frac{0^2}{2} = \frac{15\tau x(0)}{\rho U} + C \rightarrow C=0$$

$$\delta = 5477 \sqrt{\frac{x}{U}}$$

The boundary layer thickness can be expressed in dimensional form as:

$$\frac{\delta}{x} = \frac{5477}{x} \sqrt{\frac{x}{U}} \rightarrow \frac{\delta}{x} = \frac{5477}{\sqrt{Re}}$$

Kármán-Pohlhausen approximation (cubic)

$$\frac{u}{U} = a + b \left(\frac{y}{\delta} \right) + c \left(\frac{y}{\delta} \right)^2 + d \left(\frac{y}{\delta} \right)^3$$

• $u=0$ at $y=0$ → From x-momentum equation particularized in $y=0$

$$\bullet \frac{\partial^2 u}{\partial y^2} \text{ at } y=0 \rightarrow u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \frac{\partial^2 u}{\partial y^2} \rightarrow \frac{\partial^2 u}{\partial y^2} = 0(y=0)$$

• $u=U$ at $y=\delta$ → Pressure does not depend on "x"

$$\bullet \frac{\partial u}{\partial y} = 0 \text{ at } y=\delta \rightarrow \frac{\partial u}{\partial y} = 0$$

B.C adimensionalization

$$\bullet \frac{u}{U} = 0 \text{ at } \frac{y}{\delta} = 0 \rightarrow 0 = a + b(0) + c(0) + d(0) \rightarrow [a = 0]$$

$$\bullet \frac{\partial^2 (\frac{u}{U})}{\partial (\frac{y}{\delta})^2} \text{ at } \frac{y}{\delta} = 0 \rightarrow 0 = 2c + 6d(0) \rightarrow [c = 0]$$

$$\bullet \frac{u}{U} = 1 \text{ at } \frac{y}{\delta} = 1 \rightarrow 1 = b(1) + d(1)^3 \quad \begin{cases} [d = -\frac{1}{2}] \\ [b = \frac{3}{2}] \end{cases}$$

$$\bullet \frac{\partial (\frac{u}{U})}{\partial (\frac{y}{\delta})} = 0 \text{ at } \frac{y}{\delta} = 1 \rightarrow 0 = b + 3d(1)$$

$$\frac{u}{U} = \frac{3}{2} \left(\frac{y}{\delta}\right) - \frac{1}{2} \left(\frac{y}{\delta}\right)^3$$

Momentum integral equation (flat plate)

$$\frac{d}{dx} \int_0^{\delta} \left[\frac{3}{2} \left(\frac{y}{\delta}\right) - \frac{1}{2} \left(\frac{y}{\delta}\right)^3 \right] \left[1 - \frac{3}{2} \left(\frac{y}{\delta}\right) + \frac{1}{2} \left(\frac{y}{\delta}\right)^3 \right] dy = \frac{\tau_0}{\rho U^2}$$

$$\frac{d}{dx} \left[\frac{3}{2} \cdot \frac{y^2}{2\delta} - \frac{9}{4} \cdot \frac{y^3}{3\delta^2} + \frac{3}{4} \cdot \frac{y^5}{5\delta^4} - \frac{1}{2} \cdot \frac{y^4}{4\delta^3} + \frac{3}{4} \cdot \frac{y^5}{5\delta^3} - \frac{1}{4} \cdot \frac{y^7}{7\delta^6} \right] = \frac{\tau_0}{\rho U^2}$$

$$\frac{d}{dx} \left(\frac{39\delta}{280} \right) = \frac{\tau_0}{\rho U^2} \quad \tau_0 = \mu \cdot \frac{U}{\delta} \left(\frac{3}{2} (1 - (\frac{y}{\delta})^2) \right) = \mu \cdot \frac{U}{\delta} \cdot \frac{3}{2}$$

$$\frac{d}{dx} \left(\frac{39\delta}{280} \right) = \frac{3\mu}{2\rho U^2} \rightarrow \boxed{\frac{d\delta}{dx} = \frac{140}{13} \cdot \frac{U}{\delta}} \quad \text{ODE}$$

Resolution.

$$\frac{d\delta}{dx} = \frac{140}{13} \cdot \frac{U}{\delta} \rightarrow \delta d\delta = \frac{140}{13} \cdot \frac{U}{\delta} dx \rightarrow \int \delta d\delta = \int \frac{140}{13} \cdot \frac{U}{\delta} dx \rightarrow$$

$$\rightarrow \frac{\delta^2}{2} = \frac{140}{13} \cdot \frac{U}{\delta} x + C \rightarrow \delta^2 = \frac{280}{13} \cdot \frac{U}{\delta} x \rightarrow \delta = 4'641 \sqrt{\frac{Ux}{13}}$$

$$\delta(0) = 0 \rightarrow 0^2 = \frac{140}{13} \cdot \frac{U}{\delta}(0) + C \rightarrow C = 0$$

The boundary layer thickness is expressed in adimensional form as:

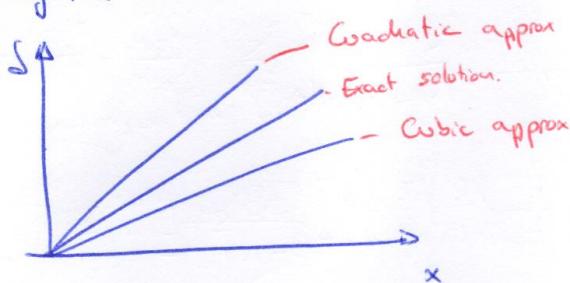
$$\frac{\delta}{x} = \frac{4'641}{x} \sqrt{\frac{Ux}{13}} \rightarrow \boxed{\frac{\delta}{x} = \frac{4'641}{\sqrt{Re}}}$$

Blasius exact solution

$$\frac{\delta}{x} = \frac{5}{\sqrt{Re}}$$

Comparing the previous approximations to the exact solution we can observe that the cubic approximation is closer, as expected.

Plot of the solutions:



As can be seen in the schematic representation, the cubic approximation gives a thinner boundary ~~thickness~~ layer regarding to the exact solution, whereas the quadratic approximation gives a wider one.